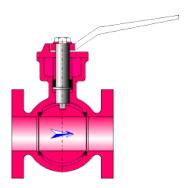


# Ball valve (Manufacturer defined)



# Model description:

This model of component calculates the minor head loss (pressure drop) generated by the flow in a ball valve installed in a straight pipe.

The valve characteristics are defined by valves manufacturers. The pressure drop of the valve is characterized by a flow coefficient "Kv", "Cv" or "Av" at full opening, and a law of flow coefficient evolution according the valve opening.

#### Model formulation:

Cross-sectional area  $(m^2)$ :

$$A = \pi \cdot \frac{D^2}{4}$$

Mean velocity (m/s):

$$U = \frac{Q}{A}$$

Mass flow rate (kg/s):

$$G = Q \cdot \rho$$

Reynolds number:

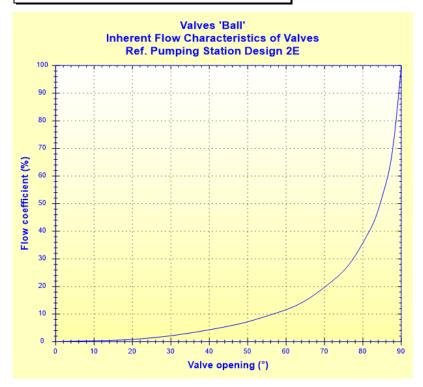
$$Re = \frac{U \cdot D}{v}$$

Evolution of the flow coefficients according to the valve opening:

$$Kv = Kvs \cdot F_{low}C_{oef}(V_{alve}O_{pening})/100$$

$$Cv = Cvs \cdot F_{low}C_{oef} (V_{alve}O_{pening})/100$$

$$Av = Avs \cdot F_{low}C_{oef}(V_{alve}O_{pening})/100$$



 $F_{low}C_{oef}(V_{alve}O_{pening})$  ([1]

Figure 5-2)

Local resistance coefficient (Pa):

■  $Re \ge 10^4$  (turbulent flow)

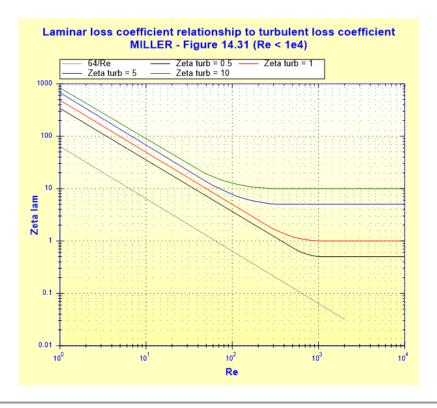
$$K_{turb} = \frac{2 \cdot A^2}{\left(\frac{Kv}{36023}\right)^2}$$

$$K_{turb} = \frac{2 \cdot A^2}{\left(\frac{Cv}{41650}\right)^2}$$

$$K_{turb} = \frac{2 \cdot A^2}{Av^2}$$

 $\blacksquare$  Re  $< 10^4$  (laminar flow)

$$K_{lam} = f(K_{turb}, Re)$$
 ([2] figure 14.31)



Reynolds Number Correction (Re < 10<sup>4</sup>):

$$C_{\text{Re}} = \frac{K_{lam}}{K_{turb}}$$

Corrected flow coefficient in laminar flow (Re  $< 10^4$ ):

$$Kv_c = \frac{Kv}{\sqrt{C_{Re}}}$$

$$Cv_c = \frac{Cv}{\sqrt{C_{Re}}}$$

$$Av_C = \frac{Av}{\sqrt{C_{Re}}}$$

Total pressure loss coefficient (based on mean velocity):

■ turbulent flow (Re  $\geq 10^4$ ):

$$K = K_{turb}$$

■ laminar flow (Re  $< 10^4$ ):

$$K = K_{lam}$$

Total pressure loss (Pa):

$$\Delta P = K \cdot \frac{\rho \cdot U^2}{2}$$

Total head loss of fluid (m):

$$\Delta H = K \cdot \frac{U^2}{2 \cdot g}$$

## Hydraulic power loss (W):

$$Wh = \Delta P \cdot Q$$

## Symbols, Definitions, SI Units:

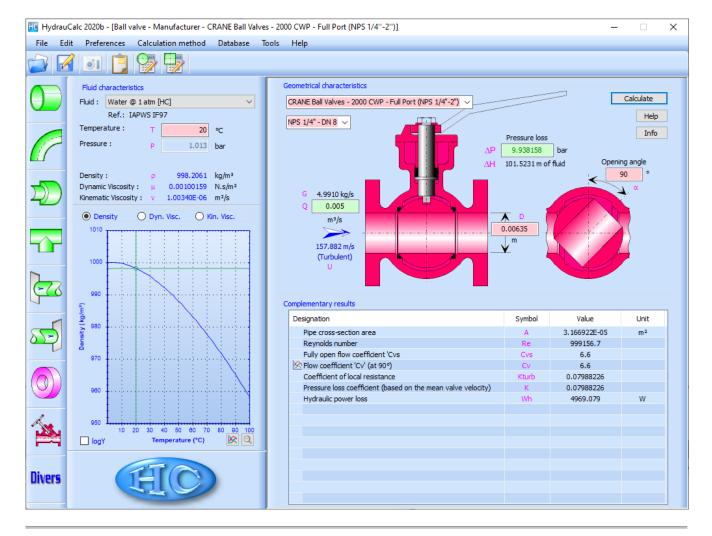
```
Internal diameter (m)
D
          Cross-sectional area (m<sup>2</sup>)
Α
Q
          Volume flow rate (m<sup>3</sup>/s)
G
          Mass flow rate (kg/s)
U
          Mean velocity (m/s)
Re
          Reynolds number ()
          Opening angle (°)
α
Kvs
          Full opening flow coefficient (m<sup>3</sup>/h)
Cvs
          Full opening flow coefficient (USG/min)
Avs
          Full opening flow coefficient (m<sup>2</sup>)
          Partial opening flow coefficient (m<sup>3</sup>/h)
Κv
Cv
          Partial opening flow coefficient (USG/min)
          Partial opening flow coefficient (m<sup>2</sup>)
Av
          Local resistance coefficient for Re \ge 10^4 ()
Kturb
          Local resistance coefficient for Re < 10<sup>4</sup> ()
Klam
          Reynolds number correction for Re < 10<sup>4</sup> ()
CRe
          Corrected flow coefficient in laminar flow (m<sup>3</sup>/h)
Kv<sub>c</sub>
          Corrected flow coefficient in laminar flow (USG/min)
Cvc
          Corrected flow coefficient in laminar flow (m<sup>2</sup>)
Av_c
Κ
          Total pressure loss coefficient (based on mean velocity) ()
\Delta \mathsf{P}
          Total pressure loss (Pa)
\Delta H
          Total head loss of fluid (m)
Wh
          Hydraulic power loss (W)
          Fluid density (kg/m³)
ρ
          Fluid kinematic viscosity (m<sup>2</sup>/s)
ν
          Gravitational acceleration (m/s^2)
q
```

# Validity range:

• any flow regime: laminar and turbulent

note: for laminar flow regime (Re  $< 10^4$ ), the pressure loss coefficient " $K_{lam}$ " is estimated

#### Example of application:



#### References:

- [1] Pumping Station Design, Second Edition, Garr M. Jones
- [2] Internal Flow System, Second Edition, D.S. Miller

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