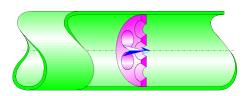


# Bevelled-Edged Grid Circular Cross-Section (IDELCHIK)



# Model description:

This model of component calculates the minor head loss (pressure drop) generated by the flow in a bevelled-edged grid (perforated plate) installed in a straight pipe.

The head loss by friction in the inlet and outlet piping is not taken into account in this component.

# Model formulation:

Hydraulic diameter (m):

$$D_h = D_0$$

Pipe cross-section area (m<sup>2</sup>):

$$F_1 = \pi \cdot \frac{D_1^2}{4}$$

Cross-section area of one hole (m<sup>2</sup>):

$$f_0 = \pi \cdot \frac{{D_0}^2}{4}$$

Clear cross-sectional area of the grid (m<sup>2</sup>):

$$F_0 = f_0 \cdot N$$

Mean velocity in pipe (m/s):

$$W_1 = \frac{Q}{F_1}$$

Mean velocity in holes (m/s):

$$W_0 = \frac{Q}{F_0}$$

Mass flow rate (kg/s):

$$G = Q \cdot \rho$$

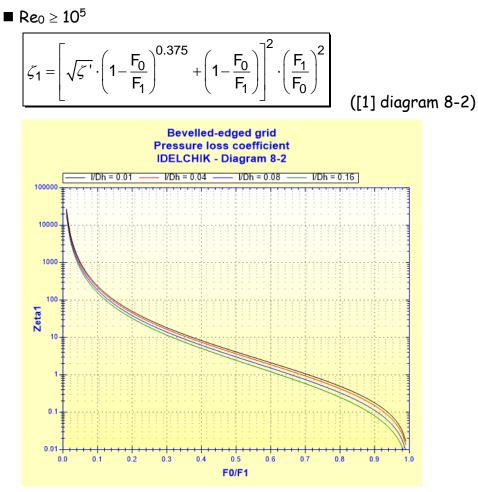
Reynolds number in pipe:

$$\mathsf{Re}_1 = \frac{W_1 \cdot D_1}{v}$$

Reynolds number in holes:

$$\mathsf{Re}_0 = \frac{\mathsf{W}_0 \cdot \mathsf{D}_0}{\mathsf{V}}$$

Local resistance coefficient:

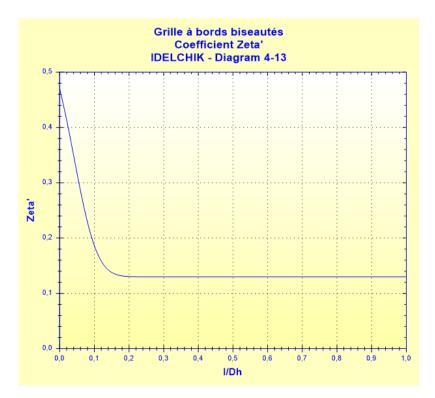


with :

Coefficient of effect of the orifice thickness:

$$\zeta' = 0.13 + 0.34 \cdot 10^{-\left(3.4 \cdot \frac{l}{D_h} + 88.4 \left(\frac{l}{D_h}\right)^{2.3}\right)}$$

([1] diagram 4-13)



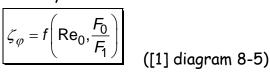
■ 
$$Re_0 \le 10^5$$

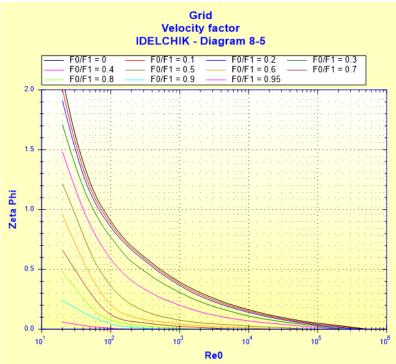
#### Quadratic local resistance coefficient:

$$\zeta_{1quad} = \left[\sqrt{\zeta'} \cdot \left(1 - \frac{F_0}{F_1}\right)^{0.375} + \left(1 - \frac{F_0}{F_1}\right)^2 \cdot \left(\frac{F_1}{F_0}\right)^2\right]$$

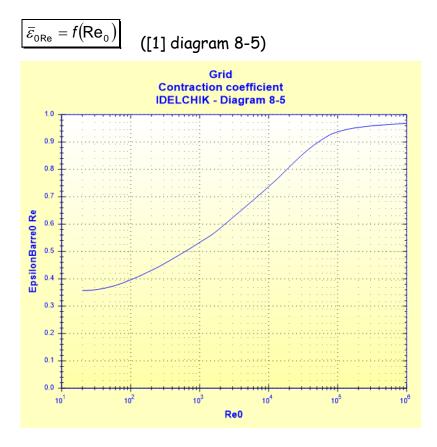
([1] diagram 8-2)

Velocity factor:





Contraction factor:



Coefficient of local resistance:

•  $30 < Re_0 < 10^5$ 

$$\zeta_{1} = \zeta_{\varphi} \cdot \left(\frac{F_{1}}{F_{0}}\right)^{2} + \overline{\varepsilon}_{0 \text{Re}} \cdot \zeta_{1 \text{quad}}$$
([1] diagram 8-5)

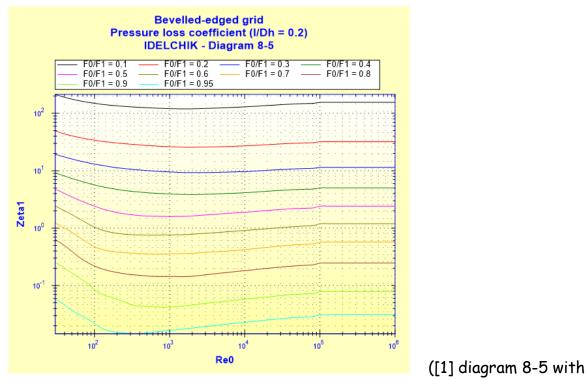
•  $10 < \text{Re}_0 \le 30$ 

$$\zeta_{1} = \frac{33}{\text{Re}_{0}} \cdot \left(\frac{F_{1}}{F_{0}}\right)^{2} + \overline{\varepsilon}_{0\text{Re}} \cdot \zeta_{1\text{quad}}$$

([1] diagram 8-5)

•  $Re_0 \le 10$ 

$$\zeta_1 = \frac{33}{\text{Re}_0} \cdot \left(\frac{F_1}{F_0}\right)^2$$
 ([1] diagram 8-5)



### I/Dh = 0.2)

Pressure loss coefficient (based on the mean pipe velocity):

$$\zeta = \zeta_1$$

Total pressure loss (Pa):

$$\Delta \boldsymbol{P} = \boldsymbol{\zeta} \cdot \frac{\boldsymbol{\rho} \cdot \boldsymbol{W}_1^2}{2}$$

Total head loss of fluid (m):

$$\Delta H = \zeta \cdot \frac{W_1^2}{2 \cdot g}$$

Hydraulic power loss (W):

 $Wh = \Delta P \cdot Q$ 

### Symbols, Definitions, SI Units:

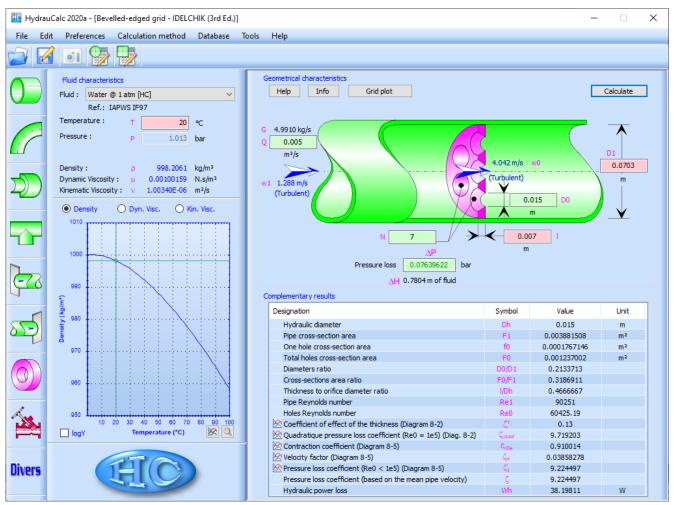
- D<sub>h</sub> Hydraulic diameter (m)
- D<sub>1</sub> Pipe internal diameter (m)
- F<sub>1</sub> Pipe cross-sectional area (m<sup>2</sup>)
- N Holes number ()
- Do Holes diameter (m)
- F<sub>0</sub> Clear cross-sectional area of the grid (m<sup>2</sup>)
- fo Cross-section area of one hole (m<sup>2</sup>)
- Q Volume flow rate (m<sup>3</sup>/s)
- w<sub>1</sub> Mean velocity in pipe (m/s)
- wo Mean velocity in holes (m/s)
- G Mass flow rate (kg/s)

I	Grid thickness (m)
Re <sub>1</sub>	Reynolds number in pipe ()
Re <sub>0</sub>	Reynolds number in holes ()
$\zeta_1$ quad	Quadratic pressure loss coefficient determined as $Re = 10^5$ ()
$\zeta_{\phi}$	Velocity factor ()
E0Re	Contraction factor ()
ζ1	Coefficient of local resistance ()
ζ	Pressure loss coefficient (based on the mean pipe velocity) ()
$\Delta P$	Total pressure loss (Pa)
ΔH	Total head loss of fluid (m)
Wh	Hydraulic power loss (W)
ρ	Fluid density (kg/m³)
ν	Fluid kinematic viscosity (m²/s)
g	Gravitational acceleration (m/s²)
-	

#### Validity range:

- any flow regime: laminar and turbulent
- stabilized flow upstream of the grid
- thickness to hole diameter ratio (I/D\_0) greater than 0.015
- top angle of the holes between 40° and 90°

#### Example of application:



# **References**:

[1] Handbook of Hydraulic Resistance, 3rd Edition, I.E. Idelchik

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